

Development of Criteria for Flameholding Tendencies within Premixer Passages for High Hydrogen Content Fuels

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LABORATORY
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UTSR Workshop

Oct 4, 2012

Contract DE-FE0007045; Joe Stoffa, Contract Monitor

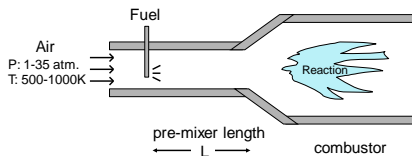
Outline

- Motivation
- Background
- Research Questions and Project Goal
- Approach and Schedule
- Test Rig Development
- Current Status

Motivation

- **Trends in Advanced Lean Burning Gas Turbines***

- Higher Combustor Inlet Temperatures
- Improved Fuel/Air Mixing
- Risk of Auto-Ignition/Flashback
- Role of Fuel Type/Composition



Major Question

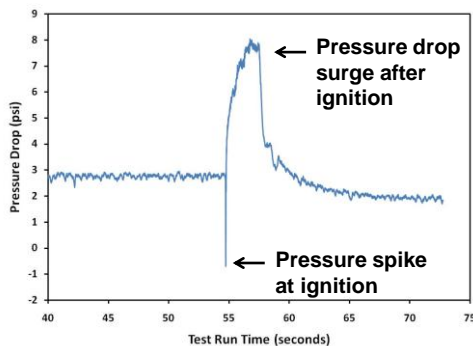
If a Reaction is Initiated in the Premixer,
Will the Reaction be "Held" on a Wall Recess?

* Stationary Gas Turbine Engines

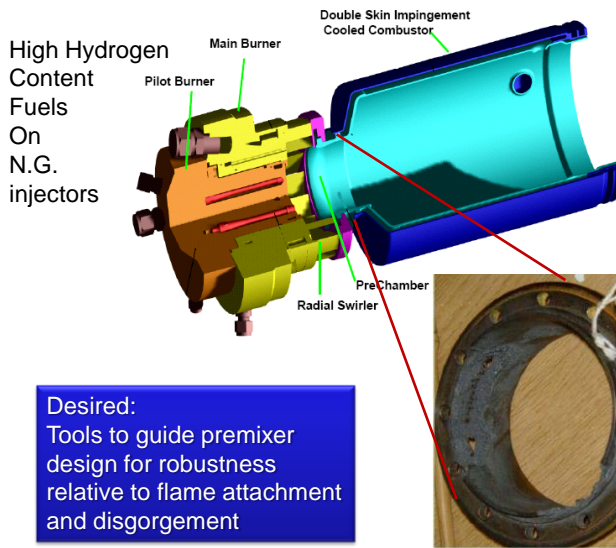


Motivation

- Ignition/reignition of all fuels (natural gas and hydrogen) leads to a pressure spike that can allow flame to propagate up into premixer
- Natural Gas reaction discharges
- Hydrogen reaction anchors in premixer zone



Motivation

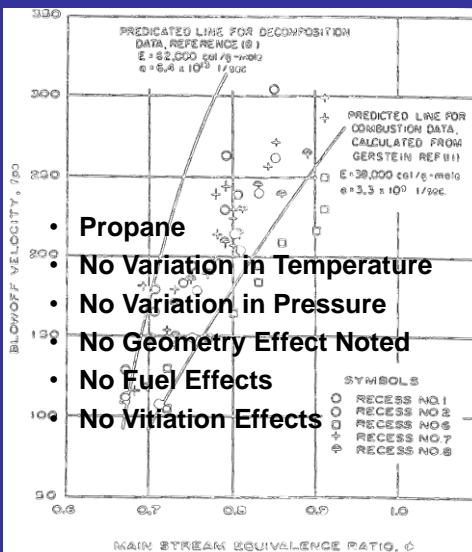


Desired:
Tools to guide premixer design for robustness relative to flame attachment and disorgement



Background

- Large Body of
- Findings
 - Only ~2
 - Most F
 - Most S
- Studies of
 - Cambe
 - Wal
 - Lim



Background

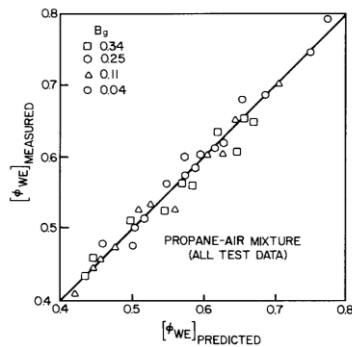
Large Body of Literature on Blowoff/Flameholding

- Findings
 - Only ~25% Focus on Natural Gas, <10% Hydrogen
 - Most Focus on Centerbody Stabilization vs. Wall Effects
 - Most Seek How to Stabilize, Not How to Avoid
- Studies of Particular Relevance
 - Cambel, et al. (1957, 1962)
 - Wall Perturbations
 - Limited Conditions
 - Cambel suggested mechanism “similar to centerbody stabilized”
 - If true....correlation work for CB Stabilized
 - Damköhler scaling seems to capture behavior
 - e.g., work of Lefebvre, others
 - e.g., Shanbhogue, Husain, and Lieuwen



Damköhler Scaling

$$\phi_{LBO} = \left\{ \frac{2.25 [1 + 0.4U(1+u')]}{P^{0.25} T_o e^{(T_o/150)} D_c (1 - B_g)} \right\}^{0.16}$$



Ballal and Lefebvre, 1979

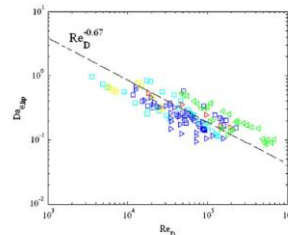
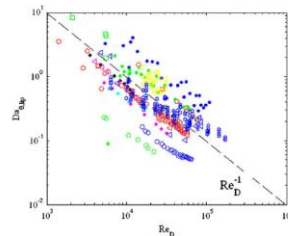


Fig. 17. Correlation of two-dimensional (top) and axisymmetric (bottom) bluff body data into showing Da_{we} at blowoff as a function of Re_D utilizing tau as the chemical time. See text and Table 1 for explanation of notations and symbols.

Shanbhogue, Husain, and Lieuwen, 2009



Research Questions

Major Question

If a Reaction is Initiated in the Premixer,
Will the Reaction be “Held” on a Wall Recess?

Related Question #1

To What Extent do “Damköhler Type” expressions (based mainly on bluff body stabilized flames) apply to “small” wall recesses and/or perturbations?

(Wall quenching/heat transfer mechanism should influence situation more so than bluff body situation)



Research Questions

Major Question

If a Reaction is Initiated in the Premixer,
Will the Reaction be “Held” on a Wall Recess?

Related Question #2

If the reaction holds on a wall feature, what is required to dislodge it (experience suggests strong hysteresis)



Research Questions

Major Question

If a Reaction is Initiated in the Premixer,
Will the Reaction be “Held” on a Wall Recess?

Related Question #3

What is role of T, P, fuel composition, and level of vitiation?



Research Questions

Major Question

If a Reaction is Initiated in the Premixer,
Will the Reaction be “Held” on a Wall Recess?

Related Question #4

How does the geometry of the wall feature affect the
flameholding tendency?



Project Goal

- **Develop design guides to predict flameholding tendencies within premixer passages as a function of:**
 - **Pressure**
 - **Temperature**
 - **Fuel Type/Composition**
 - **%O₂ in the air (vitiation levels)**
 - **Geometry Features**



Approach and Schedule

Task Description	Year 1		Year 2		Year 3	
	M1	M7	M13	M19	M25	M31
Task 1—Project Management and Planning*		*	*	*		
Task 2—Preparation -Task 2.1—Fuel/Module Selection -Task 2.2—Fabrication -Task 2.3—Diagnostics/Rig Setup -Task 2.4--Commissioning	→					
Task 3—Experimental Studies -Phase I -Phase II		↓	→			
Task 4—Analysis and Model Development				↓	↑	↓



Approach

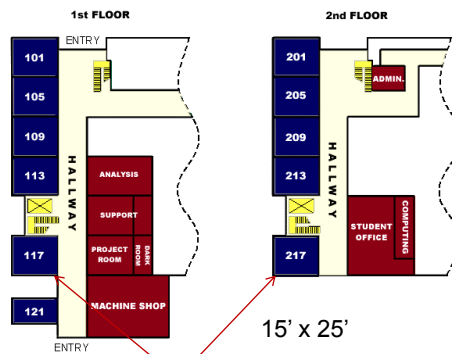
- Preparation
 - Fuel/Module Selection*
 - Fabrication
 - Diagnostics / Rig Setup
 - Commissioning
- Experimental Studies
- Analyze and Correlate Results

*Input from OEMs in early stages of project



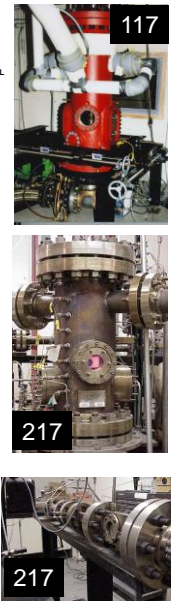
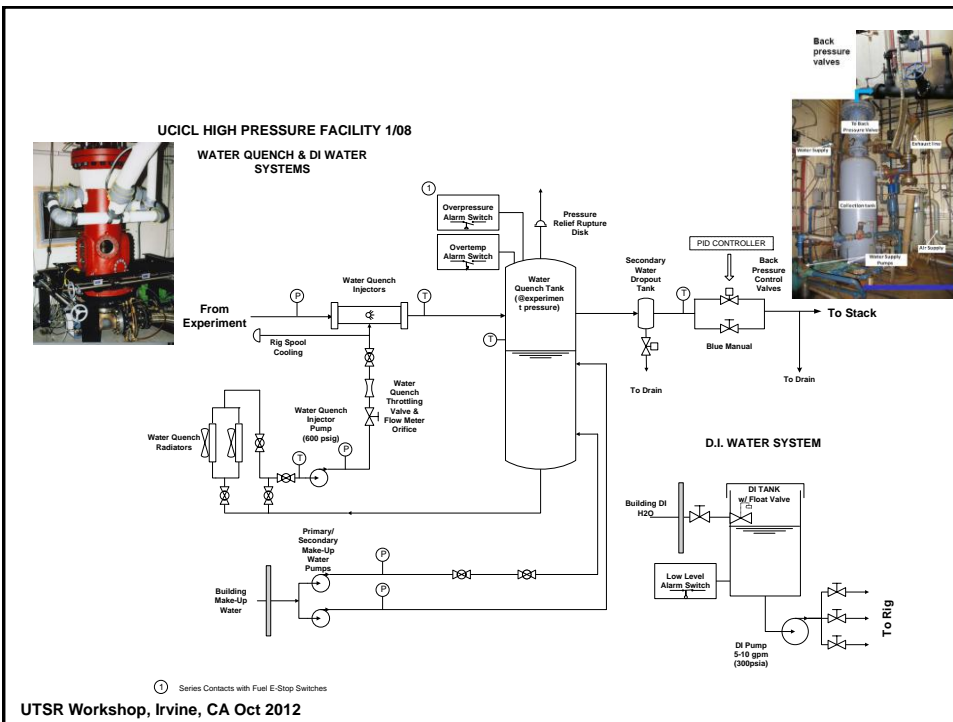
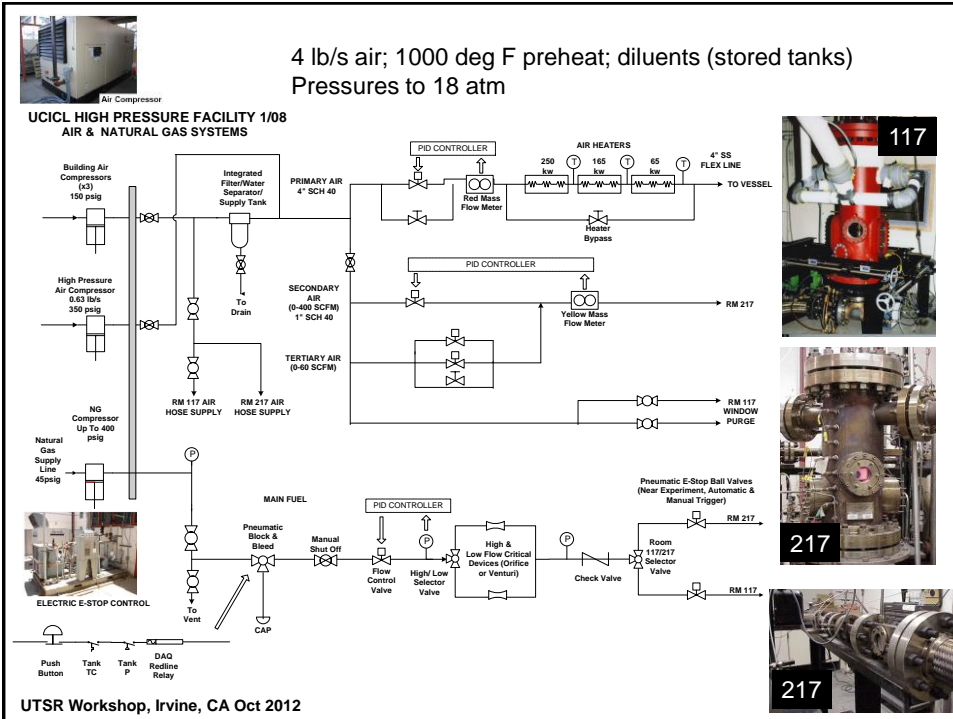
Preparation

- The test rig will leverage existing high pressure testing capability developed through support of NASA, DOE, and industry



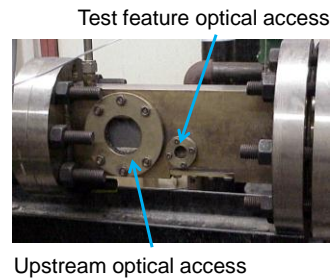
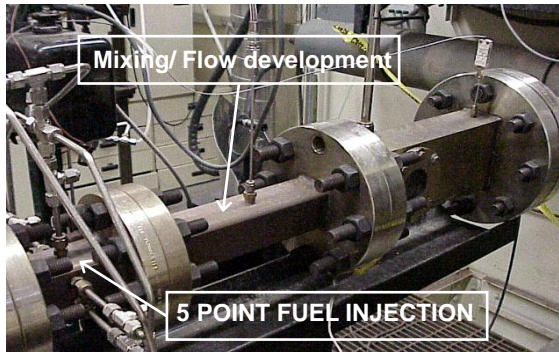
High Pressure Test Cells



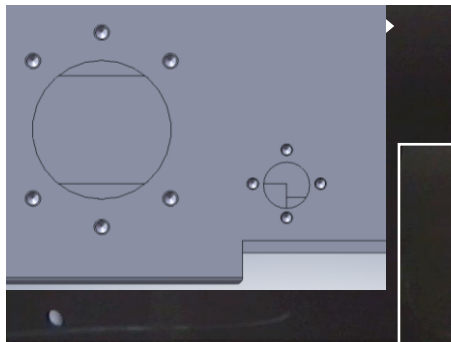


Test Rig Development

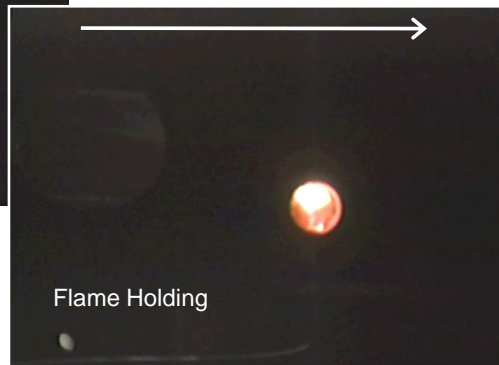
- Initial test plan envisioned using an existing "go/no go" test section



TORCH IGNITER ON



TORCH IGNITER OFF



Current Project:
High Speed OH* Imaging
will be used as well
Phantom 7.2 CMOS w/
external intensifier

Test Conditions

Test Conditions

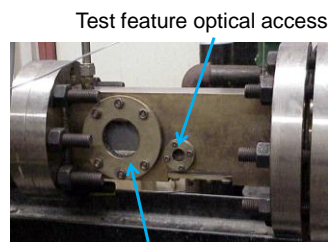
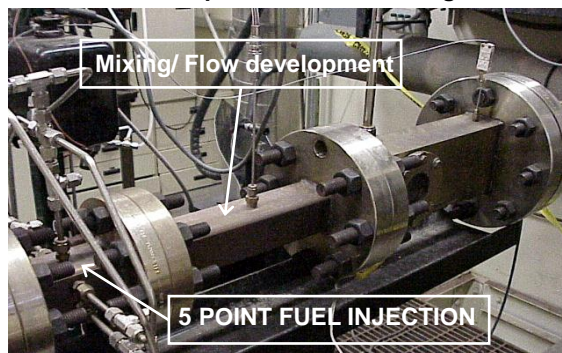
- Pressure: Up to 7 atm (*more if possible**)
- Velocity: Up to 70 m/s (*higher if possible**)
- Preheat: 500°F-1000°F (may be limited by autoignition)
- Fuel: $\Phi=0.6-1.0$
 - Hydrogen Containing Fuels (expected to be limiting case)
 - Natural Gas Fuels

*Input from OEMs in early stages of project



Test Rig Development

- Initial test plan envisioned using an existing “go/no go” test section

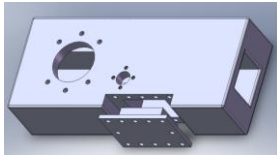


- Discussions with OEMs indicated a mismatch with desired test conditions
 - Limited to 6 atm, 60 m/s



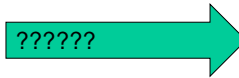
Updated Test Section

- While available test section seemed attractive.....
 - Too large to hit the desired velocities and pressures



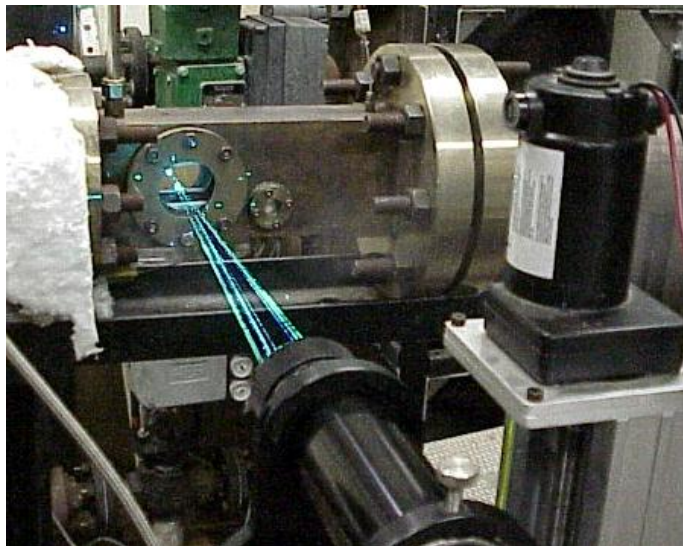
2" x 2"

??????



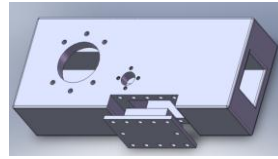
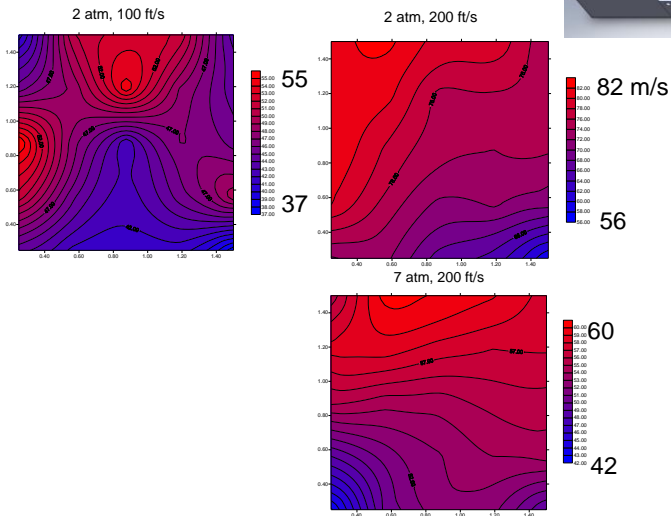
Updated Test Section

Velocity/Turbulence Mapping



Velocity/Turbulence Mapping

- Mean Velocity



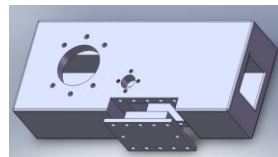
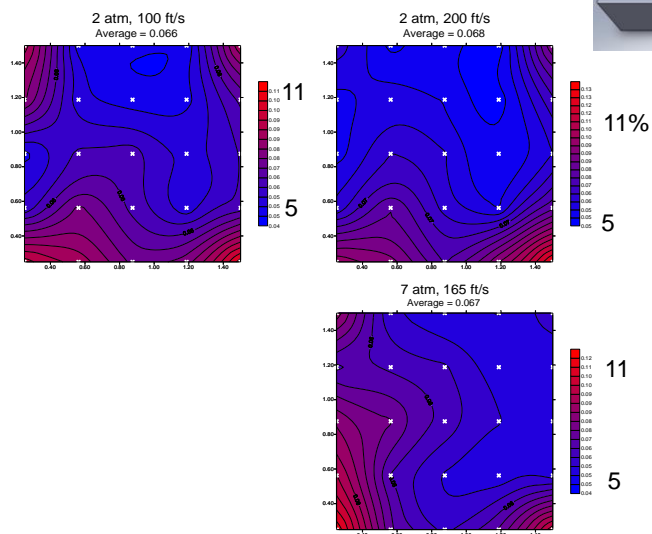
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Velocity/Turbulence Mapping

- u'/U



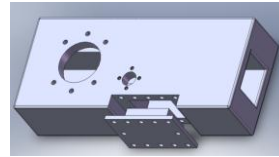
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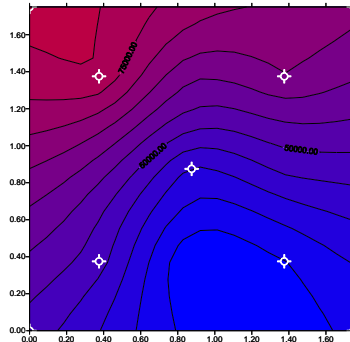


Fuel Distribution

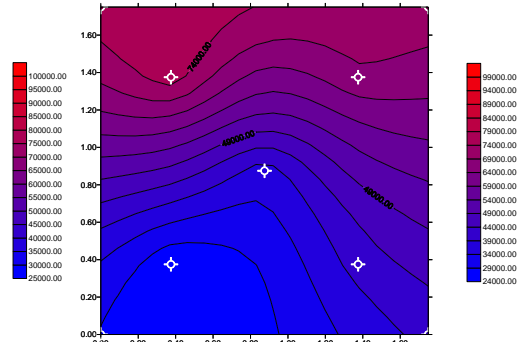
- [HC], ppm



2 atm, 100 ft/s

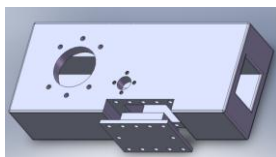


7 atm, 200 ft/s



Updated Test Section

- While available test section seemed attractive.....
 - Too large to hit the desired velocities and pressures
 - Insufficient flow conditioning/fuel distribution
 - Need to revise injection strategy



2' x 2'



Updated Test Section

Facility Heat Rejection = f(phi, Vel, T, P)

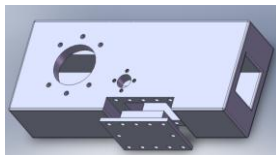
		Temperature →																
		Deg. F																
Velocity ↓	400 kW Rejection		500		600		700		800		900		1000					
			1 ATM		3 ATM		5 ATM		7 ATM		9 ATM		11 ATM		13 ATM		14 ATM	
	40	33.431	31.547	29.988	28.676	27.557	26.592	25.752	25.022	24.383	23.833	23.368	22.974	22.640	22.358	22.121	21.921	21.754
	50	41.789	39.434	37.485	35.845	34.447	33.24	32.192	31.283	30.499	29.823	29.253	28.778	28.388	28.071	27.817	27.601	27.414
	60	50.147	47.321	44.982	43.014	41.336	39.888	38.631	37.533	36.574	35.733	34.991	34.338	33.765	33.261	32.817	32.424	32.081
	70	58.505	55.208	52.479	50.183	48.226	46.536	45.079	43.822	42.733	41.783	40.961	40.258	39.665	39.171	38.767	38.444	38.191
	80	66.862	63.094	59.976	57.353	55.115	53.184	51.511	49.974	48.553	47.233	46.001	44.848	43.765	42.741	41.777	40.874	40.021
	90	75.22	70.981	67.473	64.522	62.004	59.832	57.911	56.211	54.711	53.383	52.117	50.904	49.733	48.604	47.517	46.464	45.444
	100	83.578	78.868	74.97	71.691	68.894	66.48	64.333	62.411	60.691	59.141	57.741	56.474	55.231	54.011	52.811	51.631	50.474
	40	334.31	315.47	299.88	286.76	275.57	265.92	257.52	249.99	243.11	236.83	231.11	225.92	221.17	216.83	212.89	209.31	206.07
	50	417.89	394.34	374.85	358.45	344.47	332.4	321.92	312.83	304.99	298.23	292.53	287.78	283.88	280.71	278.17	276.01	274.14
	60	501.47	473.21	448.82	430.14	413.36	398.88	385.31	372.53	360.44	349.01	338.23	328.01	318.33	309.17	300.51	292.34	284.57
	70	585.05	552.08	524.79	501.83	482.26	465.36	449.91	435.82	422.91	411.01	400.11	390.11	381.01	372.71	365.11	358.21	351.91
	80	668.62	630.94	599.76	575.53	551.15	531.84	513.41	495.74	478.81	462.51	446.81	431.71	417.11	403.01	389.41	376.31	363.71
	90	752.2	709.81	674.73	645.22	620.04	598.32	577.11	556.31	535.91	515.91	496.31	477.11	458.31	439.71	421.51	403.71	386.31
	100	835.78	788.68	749.7	716.91	688.94	664.8	641.11	617.71	594.61	571.81	549.31	527.11	505.31	483.91	462.91	442.31	422.11

Managing required heat rejection:
 Actual run times not expected to be long enough to require full heat rejection



Updated Test Section

- While available test section seemed attractive.....
 - Too large to hit the desired velocities and pressures



2" x 2"



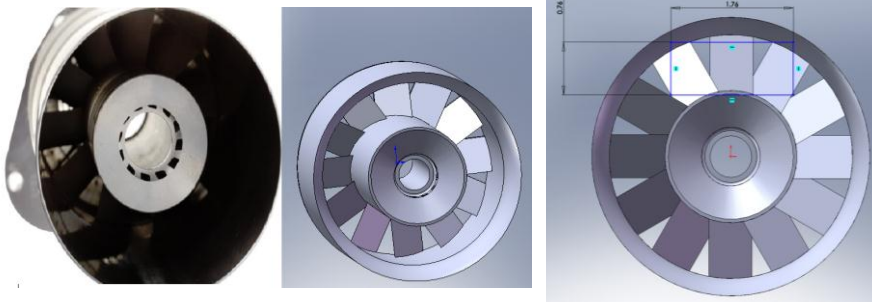
Updated Test Section
 1.76" x 0.76"
 Hits velocities at temperatures of interest while allowing sufficient heat rejection



Test Section Sizing

Is 1.76" x 0.76" cross section representative of engine premixing sections?

GE DLN



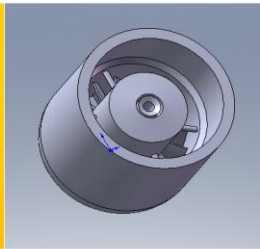
Test Section Sizing

Is 1.76" x 0.76" cross section representative of engine premixing sections?

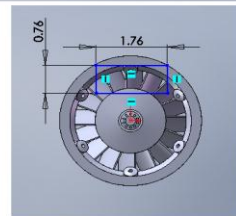
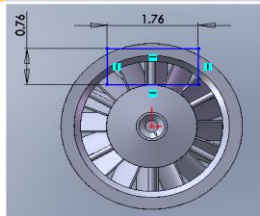
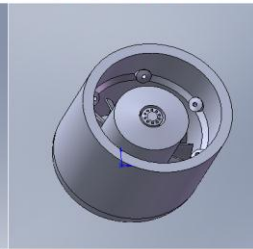
Solar Gas/Liquid Injector



Gas Injector



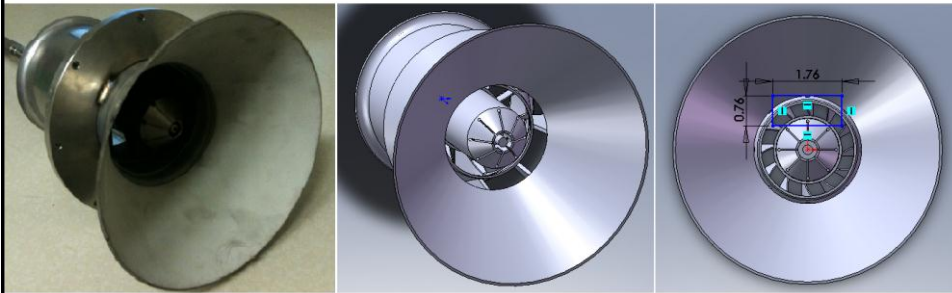
Liquid Injector



Test Section Sizing

Is 1.76" x 0.76" cross section representative of engine premixing sections?

Siemens Gas Injector/Premixer



Updated Test Section

- While available test section seemed attractive.....
 - Too large to hit the desired velocities and pressures
 - Insufficient flow conditioning/fuel distribution
 - Need to revise injection strategy

Flow Development

- Previous test section had poor velocity and fuel distributions. Attributed to short entrance length.
- The proper entrance length is calculated as:

Entrance Length Calculation: $L_E = 4.4DR_e^{3/6}$ $Re = \frac{\rho VD}{\mu} = \frac{\dot{m}D}{\mu A}$

For rectangular pipes use hydraulic diameter: $D_H = \frac{4A}{P}$

Where P is the perimeter: $2(L+W)$. Reynolds number is then: $Re = \frac{4\dot{m}}{\mu P}$

Entrance length is: $L_E = 4.4 \cdot \frac{4A}{P} \cdot \left(\frac{4\dot{m}}{\mu P}\right)^{3/6} = 4.4 \left(\frac{4LW}{2(L+W)}\right) \left(\frac{4\dot{m}}{\mu P}\right)^{3/6} = 8.8 \frac{LW}{L+W} \left(\frac{2\dot{m}}{\mu(L+W)}\right)^{3/6}$

Substituting in dimensions:

$L = 1.76'' = .0447m$

$W = 0.76'' = 0.0193m$

$\dot{m} = 1.3 \text{ lb/s} = 0.59 \text{ kg/s}$

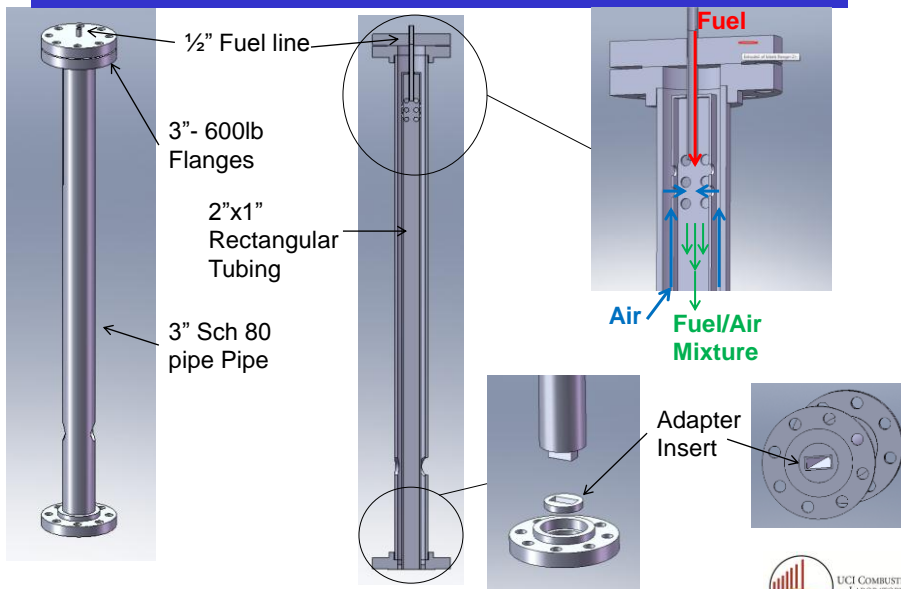
$\mu \text{ (500F)} = 2.8 \times 10^{-5} \text{ Ns/m}^2$

$$L_E = 8.8 \frac{(.0447)(.0193)}{.0447 + .0193} \left(\frac{2(0.59)}{2.8 \cdot 10^{-5}(.0447 + .0193)}\right)^{3/6} = (0.1186)(6.58 \cdot 10^{-5})^{3/6}$$

$L_E = 1.11m = 44''$

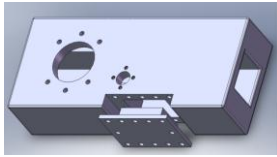


Flow Development

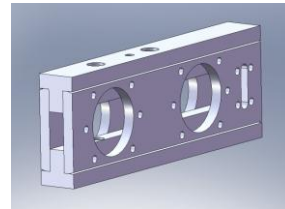


Updated Test Section

- While available test section seemed attractive.....
 - Too large to hit the desired velocities and pressures
 - Insufficient flow conditioning/fuel distribution
 - Need to revise injection strategy
 - Limited Optical Access??

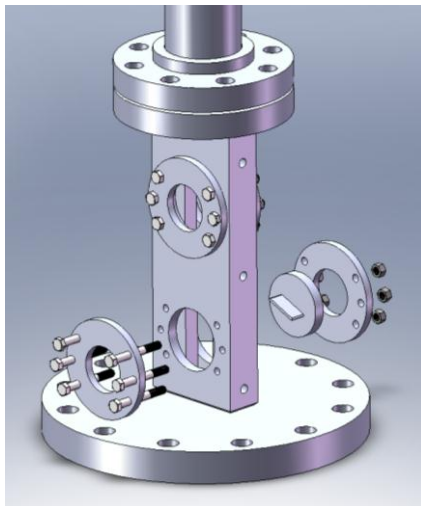


2" x 2"



1.76" x 0.76"

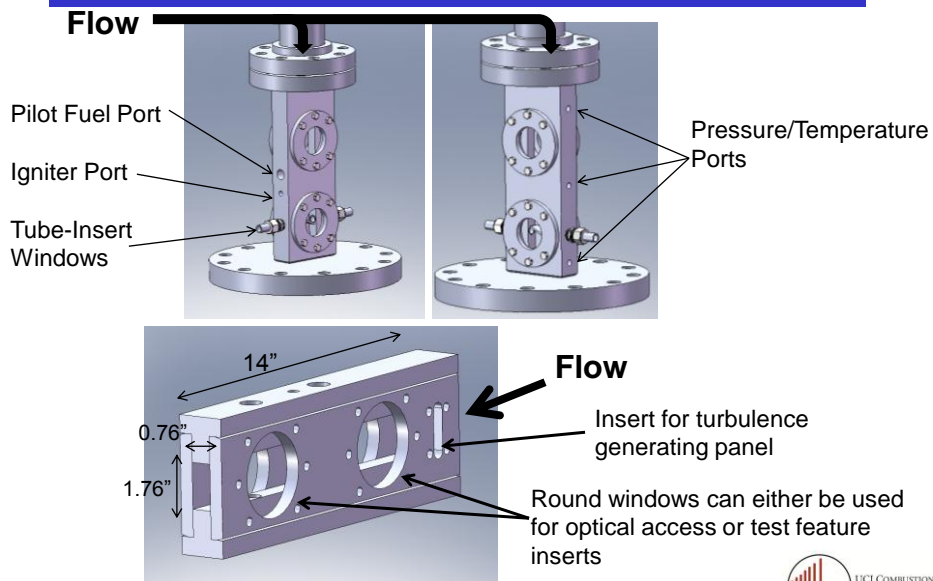
Updated Test Section



Optical windows and test feature have same base so that the test feature can be moved upstream or downstream of ignition source

Round base of test feature insert allows test feature to be rotated

Updated Test Section

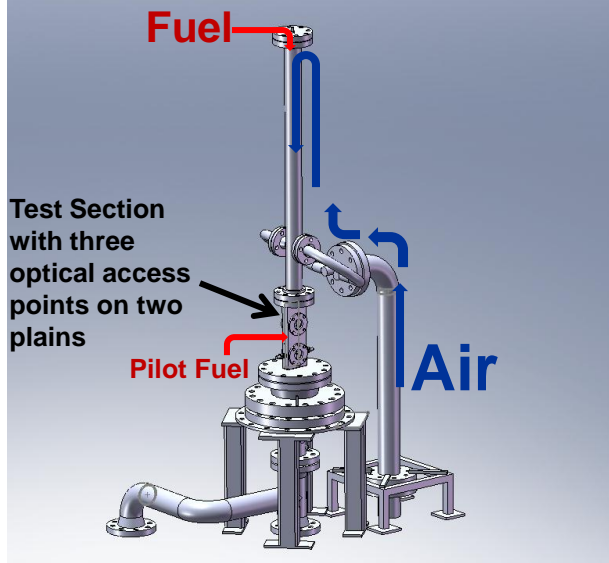


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Test Rig Assembly

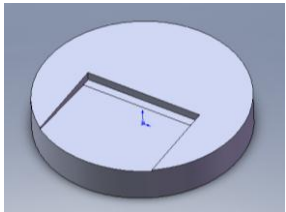


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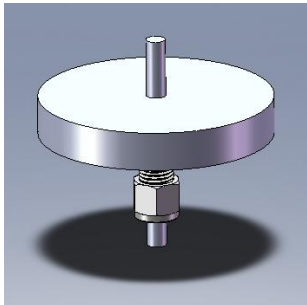
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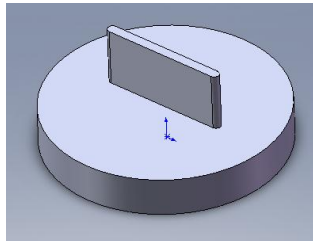
Initial Test Features*



Reverse Step



Rivet/bolt-exposed length can be varied by loosening swage fitting on underside



Vane/Strut- Feature can be rotated

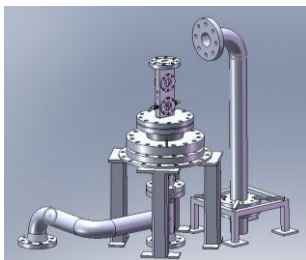
* Based on OEM Input

Status

Test Section Development: 80% Complete

Test Facility Interfacing: 60% Complete

Ready For Commissioning: end of 2012



Flameholder Test Section



Flameholder Flow Developer



Status

Task Description	Year 1		Year 2		Year 3	
	M1	M7	M13	M19	M25	M31
Task 1—Project Management and Planning*		*	*	*		
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Task 3—Experimental Studies -Phase I -Phase II						
Task 4—Analysis and Model Development						

mcdonell@ucicl.uci.edu; 949 824 5950 x121

